Dunsheath & Pemberton

An investigation of the effect of keyways on the torsional strength of shafting

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AN INVESTIGATION OF THE EFFECT OF KEYWAYS ON THE TORSIONAL STRENGTH OF SHAFTING

 \mathbf{BY}

LEROY MORRELL DUNSHEATH CARLYSLE PEMBERTON

THESIS FOR THE DEGREE OF BACHELOR OF SCIENCE

IN MECHANICAL ENGINEERING

IN THE

COLLEGE OF ENGINEERING

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190

THIS IS TO CERTIFY THAT THE THESIS PREPARED UNDER MY SUPERVISION BY

Leroy Morrell Dunsheath and Carlysle Pemberton

An Investigation of the Effect of Keyways on the **ENTITLED**

Torsional Strength of Shafting

IS APPROVED BY ME AS FULFILLING THIS PART OF THE REQUIREMENTS FOR THE

Bachelor of Science in Mechanical Engineering DEGREE OF-

G. a. Jordenough HEAD OF DEPARTMENT OF Mechanical Ingineering 1100 80 41

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AN INVESTIGATION OF THE EFFECT OF KEYWAYS ON THE TORSIONAL STRENGTH OF SHAFTING.

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AN INVESTIGATION OF THE EFFECT OF KEYWAYS ON THE TORSIONAL STRENGTH OF SHAFTING

I. Introduction.

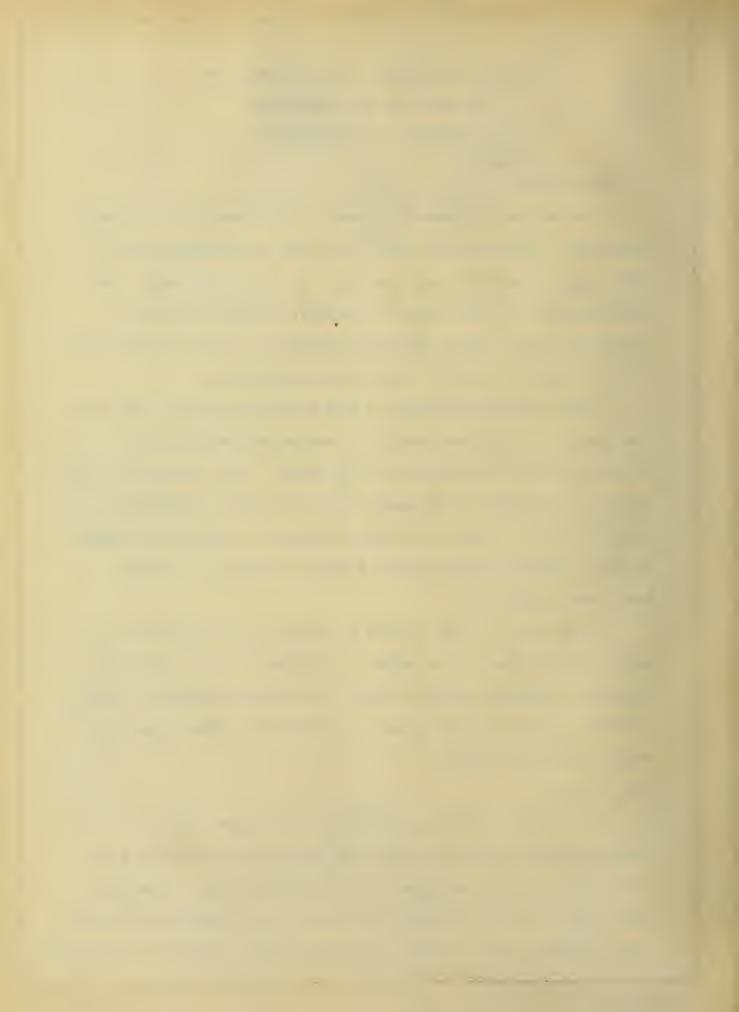
A considerable amount of data on the strength of keys is available. The several types have been standardized thru a wide range of service and loading. On the other hand, the investigation of the effect of keyways on the strength of shafts has been almost entirely neglected, no data being given on the subject in any of the various handbooks.

Some work along this line has been done, during the past two years, by the Department of Theoretical and Applied Mechanics of the University of Illinois. Two series of tests, showing the effect of keyways on the torsional strength of shafts, and one series of tests showing the effect of keyways on the combined torsional and bending strength of shafts, have been completed.

The object of this series of tests is to supplement the work already done on the effect of keyways on the torsional strength of shafts and to extend the investigations to shafts of other diameters and keyways of different forms than have heretofore been tested.

II. Theory.

The theory involved in shafts of circular cross-section under stress is fairly simple but, when the section is distorted by keyways, mathematical difficulties are introduced which have not, as yet been overcome. It is obvious, however, that a keyway, tho of small sectional area, reduces the strength



of a shaft to a marked degree, due to the fact that the metal has been removed from the portion of the section where the fiber stress is a maximum.

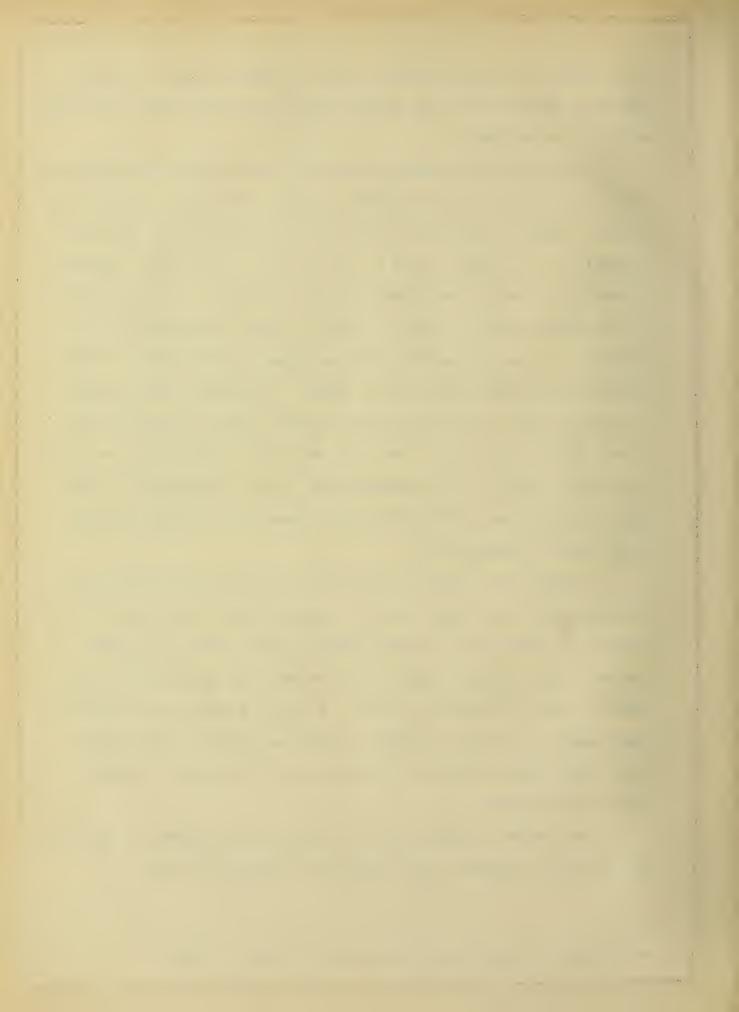
For mechanical reasons, a shaft becomes very unsatisfactory when its elastic limit has been exceeded. For this reason, the elastic limits of the shafts have been chosen as the basis of comparison. The value used is the point on the stress diagram termed by Johnson* the apparent elastic limit. It is the point on the diagram where a line, having a slope, referred to the vertical, one and one-half times as great as the slope of the stress curve below the elastic limit. The value thus obtained is greater than the value of the elastic limit as usually defined, that is the unit stress at which the increase in angular deflection ceases to be proportional to the increase in load but it is more accurately determined and serves fully as well as a basis of comparison.

In these tests the elastic limit of the various sections having keyways are expressed in terms of the elastic limit of the solid section on the same shaft, as per cent. The percentage thus obtained may then be termed the efficiency of the shaft at the particular section. A solid section was tested on each shaft in order to reduce as much as possible the errors due to the inequalities of the steel in the various shafts.

III. Test Pieces.

Three sizes of shafts were tested; cold rolled, 2" and $2\frac{1}{2}$ "; and turned, 1-15/16". The shafts were about 5' long.

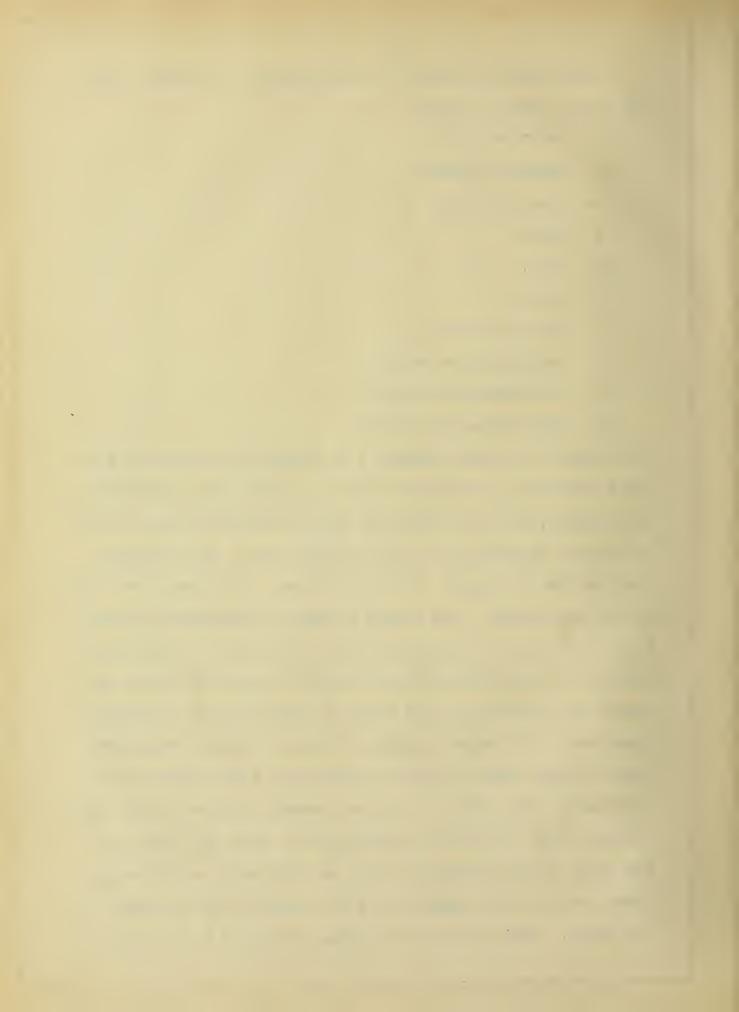
^{*} Johnson's "Materials of Construction "pp 18 -20.



The following keyways, or arrangement of keyways, were cut in the shafts tested.

- 1. Solid Section.
- 2. Standard Square.
- 3. Standard Long.
- 4. Deep.
- 5. Flat.
- 6. Round.
- 7. One Direction.
- 8. Two Standards at 90°.
- 9. Two Standards at 180°.
- 10. Four Standards at 90°.

The sketches attached (Figure 1 to Figure 22 inclusive) give the dimensions of the above for the several sizes of shafts. All keyways, with the exception of the standard long, were 7" in length, including the slope at both ends. The standard long was 24" in length, including slope. All keyways were cut to the same width. The keyway adopted as standard (Figures1, 5, 7, 8, 9, 10, 14, 18, 20, 21, and 22) is the one generally accepted in the American practice, its dimensions being one fourth the diameter of the shaft in width and one eighththe diameter of the shaft in depth. The deep keyway (Figures 6 and 19) had a depth of three sixteenths of the diameter of the shaft. The depth of the flat keyway (Figures 2, 11, and 15) was taken from "Kent's Pocket-book", page 977, table II. The round keyway (Figures 3, 12, and 16) had a semicircular cross-section, the radius being one eighth of the diameter of the shaft. The one direction keyway (Figures 4, 13, and 17)



had for the lengths of its sides one eighth, and seven thirtyseconds of the diameter of the shaft, the sides forming a right angle.

The keyways were cut at intervals of 90° around the shaft, one inch of solid stock being left between each adjacent pair. Each shaft had four keyways cut in it, with the exception of those containing the long keyway. These latter had two only. Besides these a solid section was left in each shaft.

All tests were made in duplicate. The diameter and material of the shafts were:

2" Standard Cold -rolled steel, --- Tests 1 to 6 inclusive.

1 $\frac{15}{16}$ Standard turned steel, ---- Tests 7 to 8 inclusive.

22" Standard Cold-rolled steel, ---- Tests 9 to 14 inclusive.

The shafts, listed below by their test numbers, had the follow-

Numbers 1, 2, 7, 8, 9, 10, Standard.

Flat.

One Direction.

Round.

Numbers 3, 4, 11, 12, Standard Long.

Deep.

Numbers 5, 6, 13, 14, Standard.

Two Standards at 90°.

Two Standards at 180°.

Four Standards at 90°.

IV. Apparatus.

ing keyways cut in them.

All shafts were tested on a 230,000 inch-pound Olsen
Torsion Machine in the Laboratory of Applied Mechanics of the



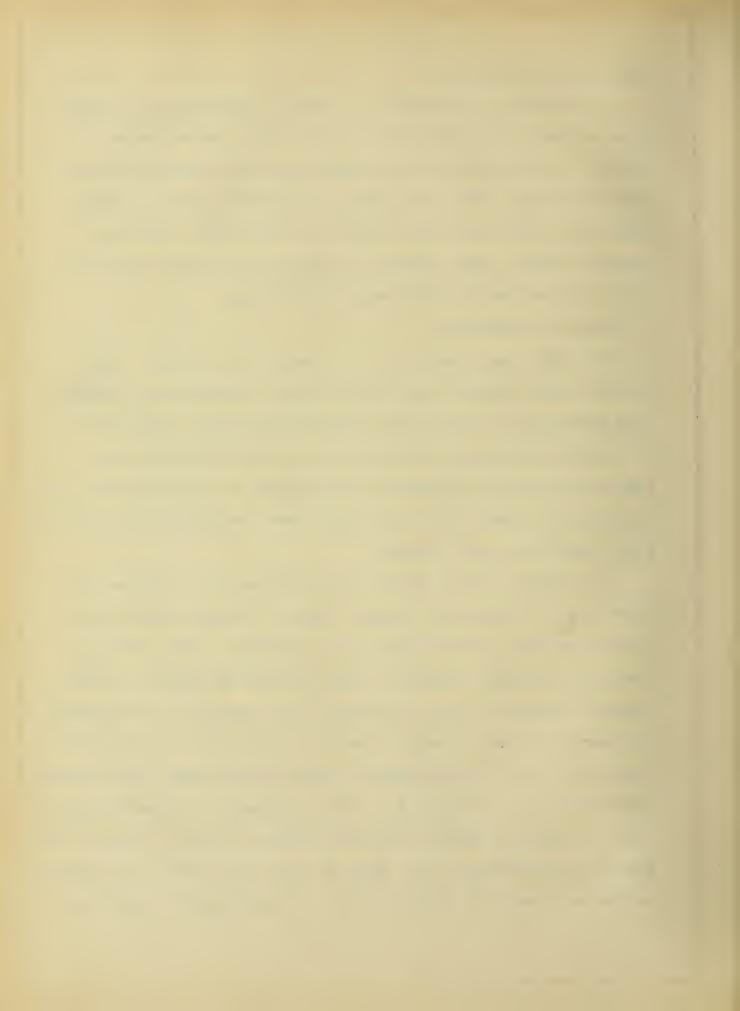
University of Illinois.

The method of indication of twisting deformation is shown by an accompanying photograph (Figure 23). The apparatus consisted of an adjustable iron clamp, carrying a wooden arm about three feet long. The iron clamp was provided with a V shaped recess and a set screw for clamping to the shaft. Mirrors, carrying scales, were attached to half of these arms while the other half carried pointers eight inches long.

V Method of Procedure.

The shaft was clamped in the jaws of the machine. Indicators were placed, eight inches apart, in the solid sections left between the keyways, an indicator bearing a scale and one bearing two pointers, alternating. It can be seen that one indicator bearing two pointers may be made to serve two sections. The adjustment was such that, when stress was applied the pointers would travel over the scales.

An initial load of 1000 inch-pounds was first put on the shaft and the indicator readings taken. The load was then increased and the various deflections recorded. After each increase the load was reduced to 1000 inch-pounds and the initial readings checked. These operations were repeated for increasing increments of load, always returning to 1000 inch-pounds between loads, until all the sections had been carried past their elastic limit. By always returning to 1000 inch-pounds and taking readings, it could be easily determined between what two loads a section took a permanent set. When it was certain that all sections had taken a set, the indicators were removed and the shaft was



stressed to rupture, the final load and approximate deflection, were recorded. Set readings were not taken for the last six tests. It was customary in this series, to approach the desired load at high speed and then use the low speed when adjusting the beam. When the load was taken off, high speed was used until the zero load was passed and then the machine was reversed and the 1000 inch-pounds approached on low speed.

VI. Derivation of Constants.

Unit Stress in Outer Fiber

S -- Unit shearing stress.

Pp -- Twisting moment in inch-pounds -- the load.

d -- Diameter of shaft in inches.

c -- Distance from axis to extreme fiber -- d/2.

J -- Polar moment of inertia for the cross-section πd4/32 for circular section (Merriman, Page 228)

 $S = Ppc/J = 16Pp/md^3 = 5.1Pp/d^3$

S = 0.637Pp for 2* shafts.

S = 0.702Pp for 1-15/16" shafts.

S = 0.326Pp for $2\frac{1}{2}$ # shafts.

Deflection in Degrees per Inch of Length.

Length of arm = 37"

Circumference of circle described by pointer

 $2\pi r = 2 \times 3.14 \times 37 = 232$

Divisions on scale are 50 to the inch.

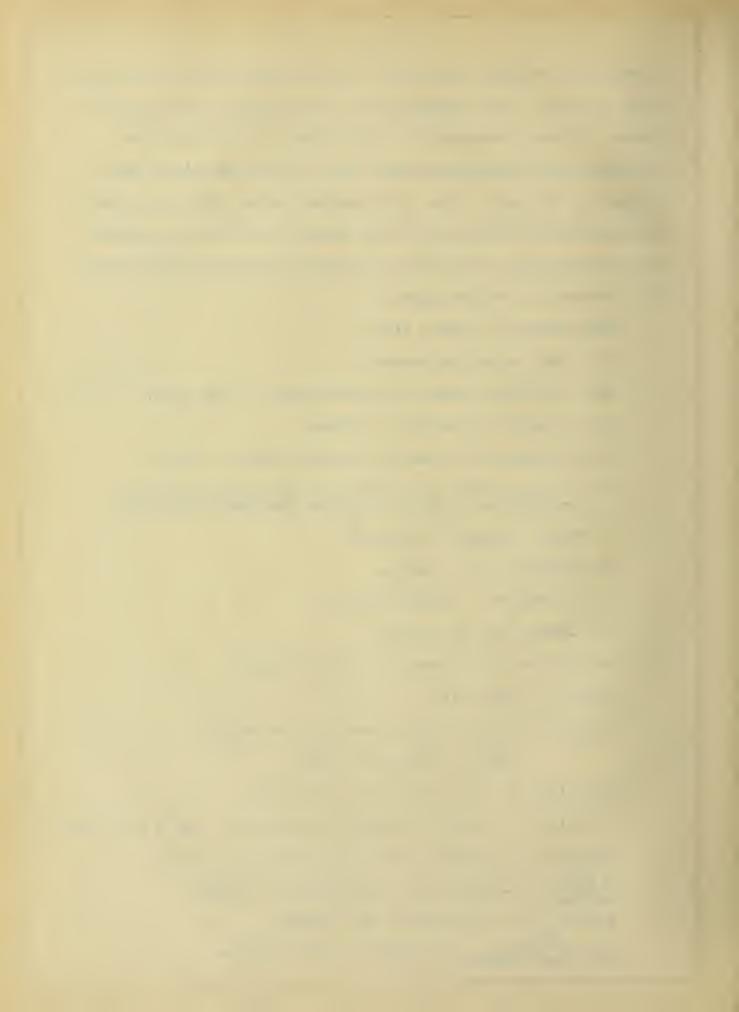
Divisions in circle described by pointer. $252 \times 50 = 11600$.

Degrees per division per inch of length of shaft.

 $\frac{360}{11600x8} = 0.00388$ for 1-15/16" and 2" shafts.

Adding $\frac{1}{4}$ " in the case of $2\frac{1}{2}$ " shafts.

 $\frac{360}{2x3.14x37.25x50x8} = 0.00386$ for $2\frac{1}{2}$ shafts.



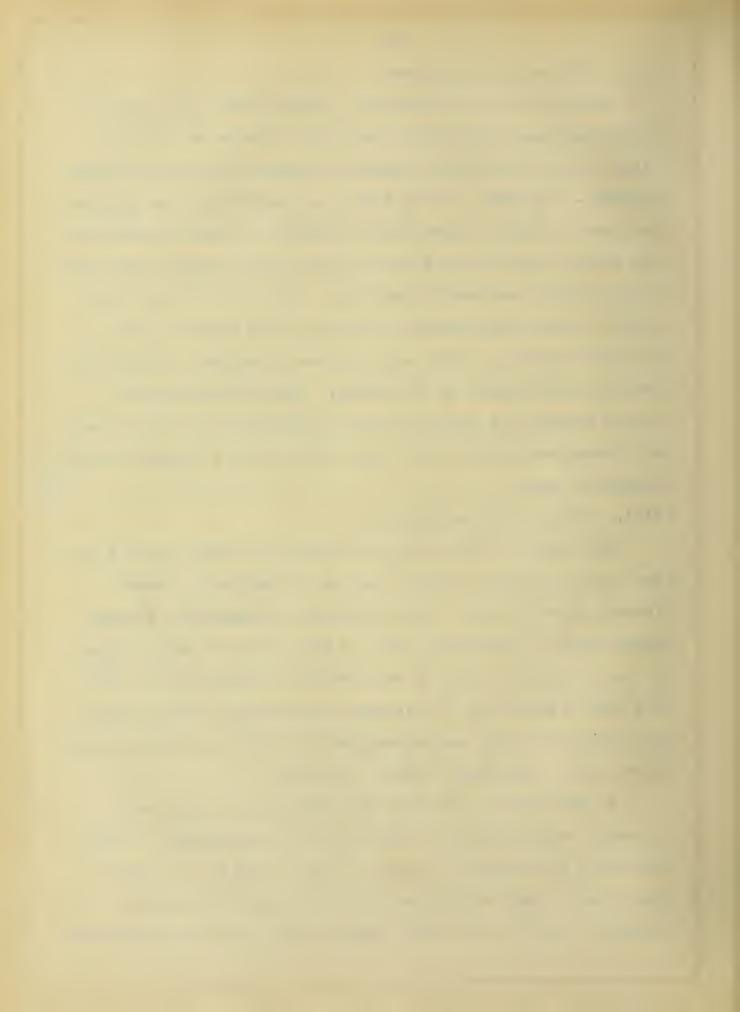
VII. Calculations and Curves.

By the aid of the constants, derived above, the load in inch-pounds was converted to unit fiber stress in the outer fiber and the deflection readings to degrees per inch of length of shaft. The data for the tests, as observed and as computed, is given in full on pages 15-29 inclusive. Stress diagrams for the various shafts were drawn and the elastic limits were found in the method previously described. The ratio of the elastic limit of the solid section on the same shaft was computed, the quotient being the efficiency of the keyway. Pages 28 and 29 contain tables showing the elastic limit and efficiency of each keyway. An average was obtained for each section with a keyway from the duplicate tests.

VIII. Results and Conclusions.

The results of the series of tests are summed up on page 29 Each test, as before stated, was run in duplicate, thereby forming check results. This, however, is inadequate for any great degree of accuracy since, if the two tests do not agree it is difficult to tell in which shaft the fault lies. It has been assumed that the average efficiencies, obtained from the duplicate tests, are approximately correct, although further tests would, obviously, reduce the error.

In the majority of cases the efficiencies check very closely, considering the simplicity of the apparatus. A considerable discrepancy is found in tests 1 and 3. In test 6 the elastic limit of the section containing two standard keyways at 180° is very low, compared with the values obtained



for the elastic limits of other similar sections. This may be due to some defect in the shaft at that place. The remainder of the results are as close as can be expected.

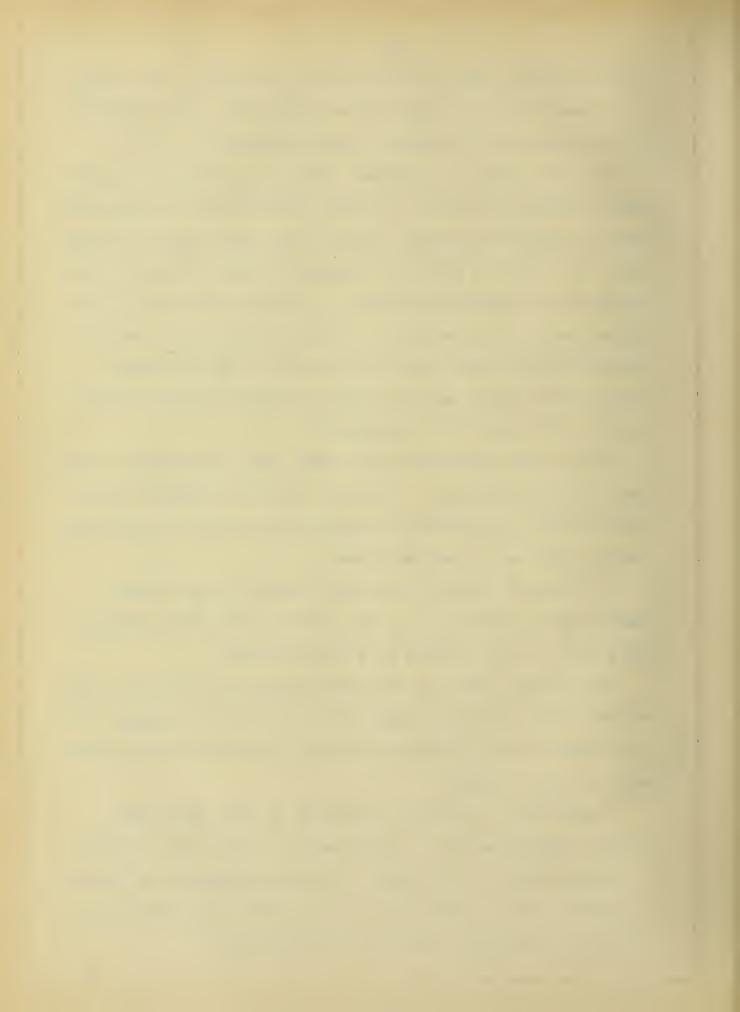
The tests show, in general, that the efficiency of shafts having keyways is greater in turned shafts than in cold-rolled shafts although the elastic limit of the solid section is much lower, and that the effect of keyways decrease in cold-rolled shafts as the diameter increases. The high efficiency in the turned shafts is probably due to the fact that the keyway in the cold-rolled shaft cuts out a section of the strongest fibers of the shaft, while the turned shafts, being of uniform quality, the effect is not so great.

It is very evident that the shaft with the keyway is weaker than the solid section. It appears that this should be taken into account in the design of shafting, especially cold-rolled shafting with more than one keyway.

The elastic limit of the solid section of the turned shafting was much lower than that of the cold rolled owing to the effect of cold rolling on the outer fiber.

The elastic limit of the solid section of the larger cold rolled shafts was higher than that of the smaller shafts. This was probably due to the fact that the cold-rolled fibers were farther from the axis.

Variations in quality of material in each shaft made accurate determinations of efficiency of shafts with keyways an impossibility. It is easy to see this variation as the last six shafts were cut from one long piece and the elastic limit of the solid portions tested vary considerably.



There is a formula derived from previous tests at the University of Illinois by Herbert F. Moore in bulletin number 42 of the University of Illinois Engineering Experiment Station, for the efficiency of cold rolled shafts.

The formula is:

e = 1.0 - 0.2w - 1.1h.

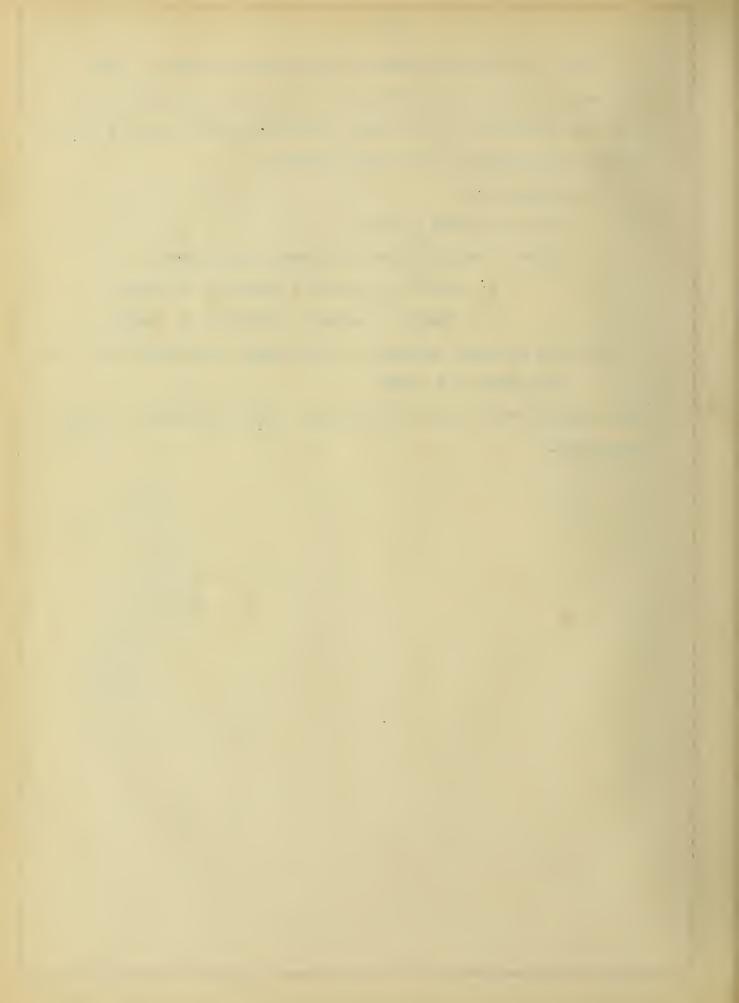
where e -- efficiency of shaft with keyway.

w -- width of keyway + diameter of shaft.

h -- depth of keyway + diameter of shaft.

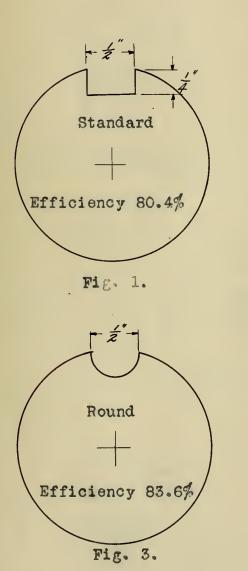
For the standard keyways in our tests, this works out to be, e=0.8125 or 81.25%

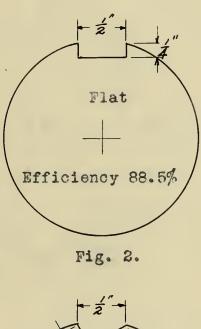
This compares very favorably with our results although it is a little low.

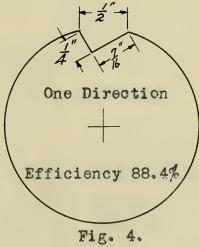


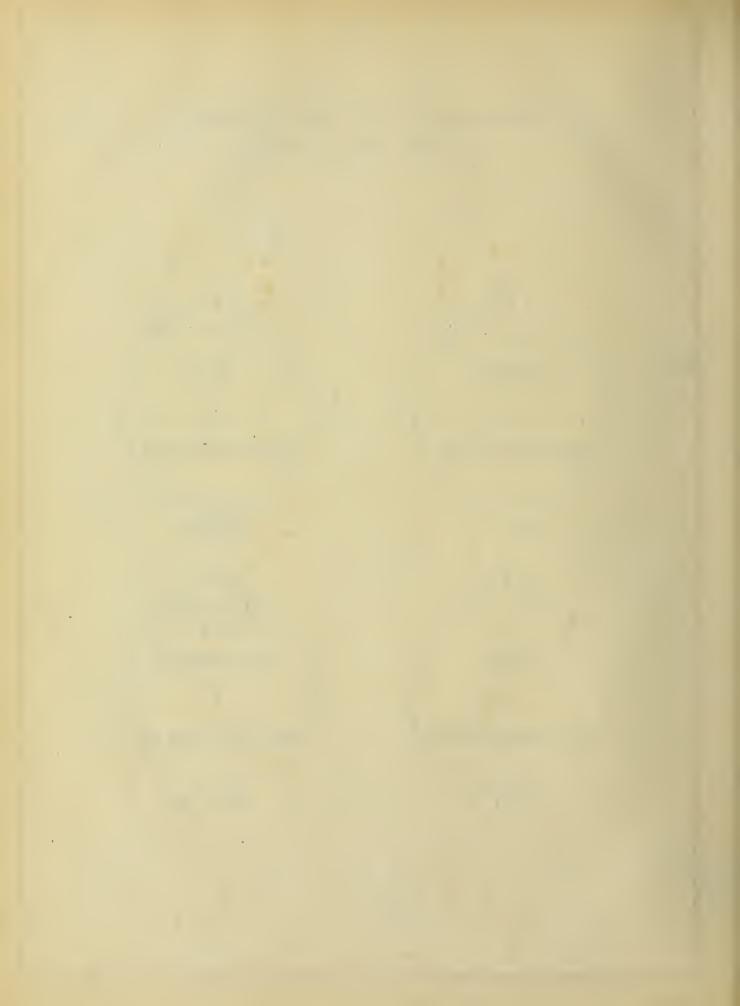
Dimensions and Efficiency of Keyways.

(2" cold rolled shafts)

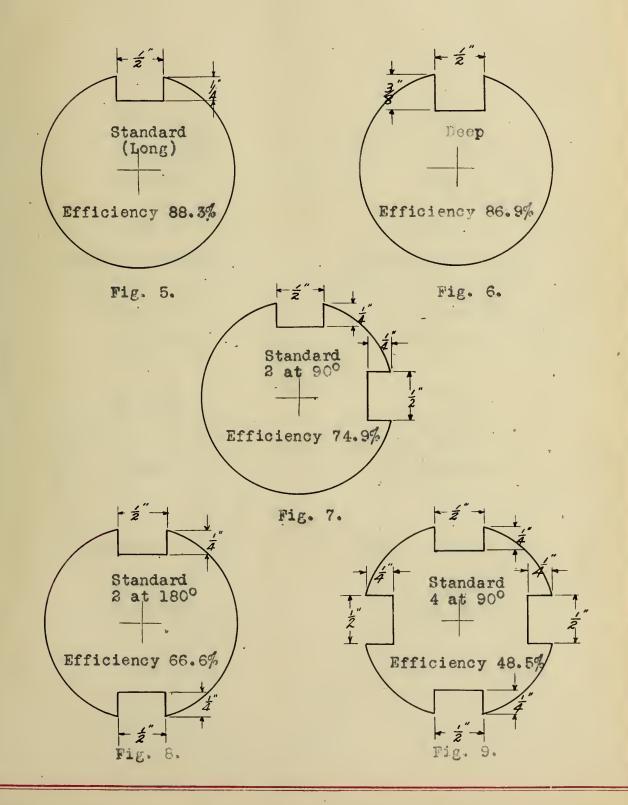


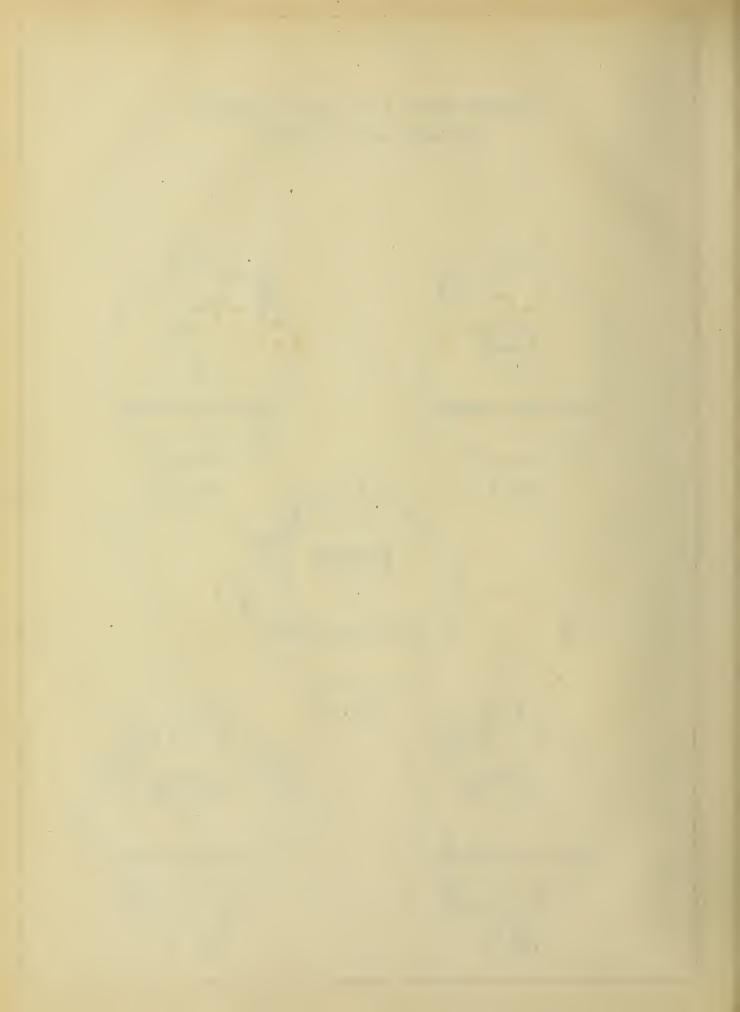






Dimensions and Efficiency of Keyways. (2" cold rolled shafts)





Dimensions and Efficiency of Keyways. (1-15/16" turned shafts)

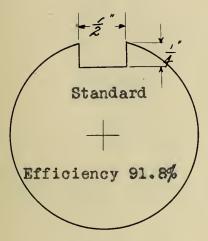
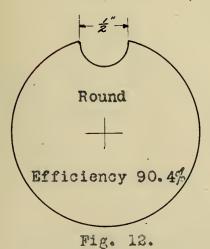


Fig. 10.



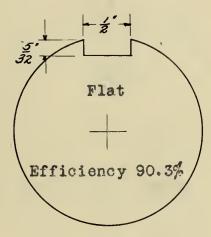


Fig. 11.

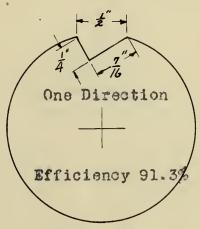
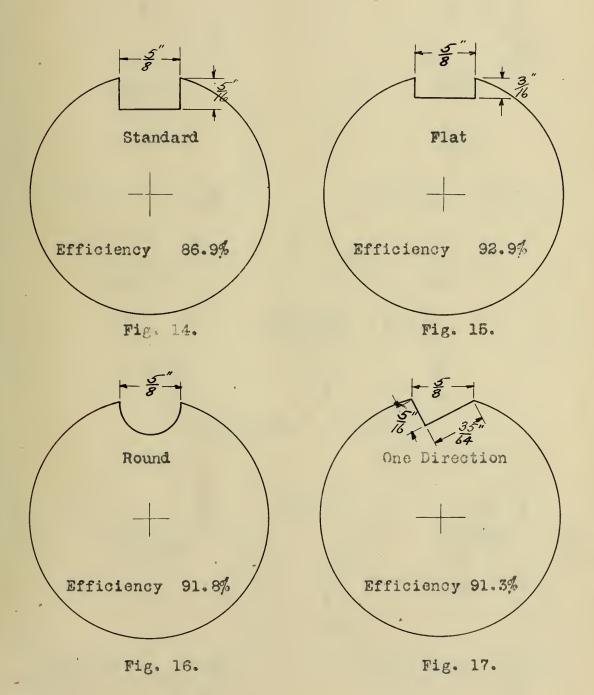


Fig. 13.



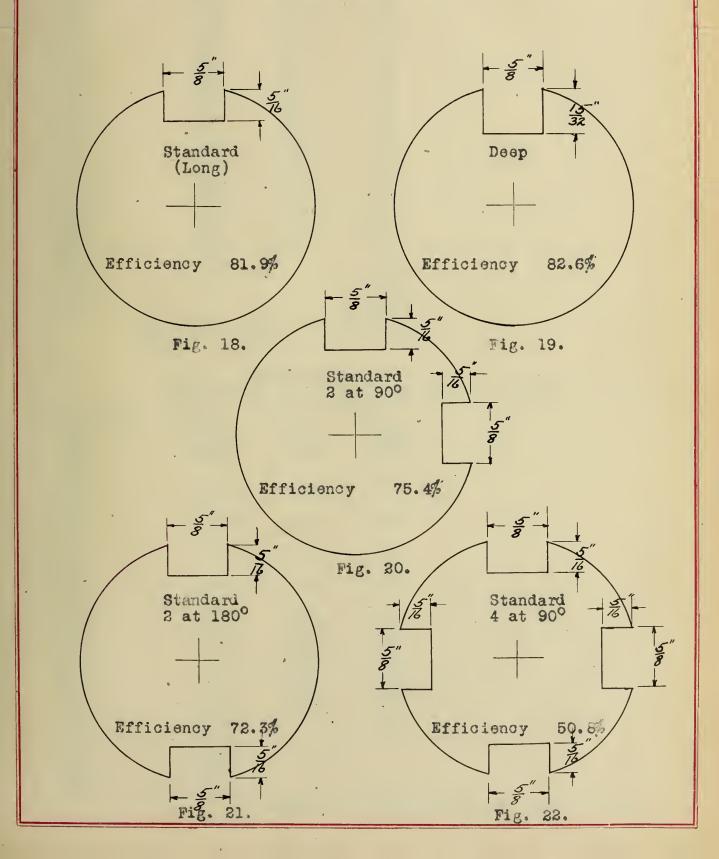
Dimensions and Efficiency of Keyways. (22 cold rolled shafts)

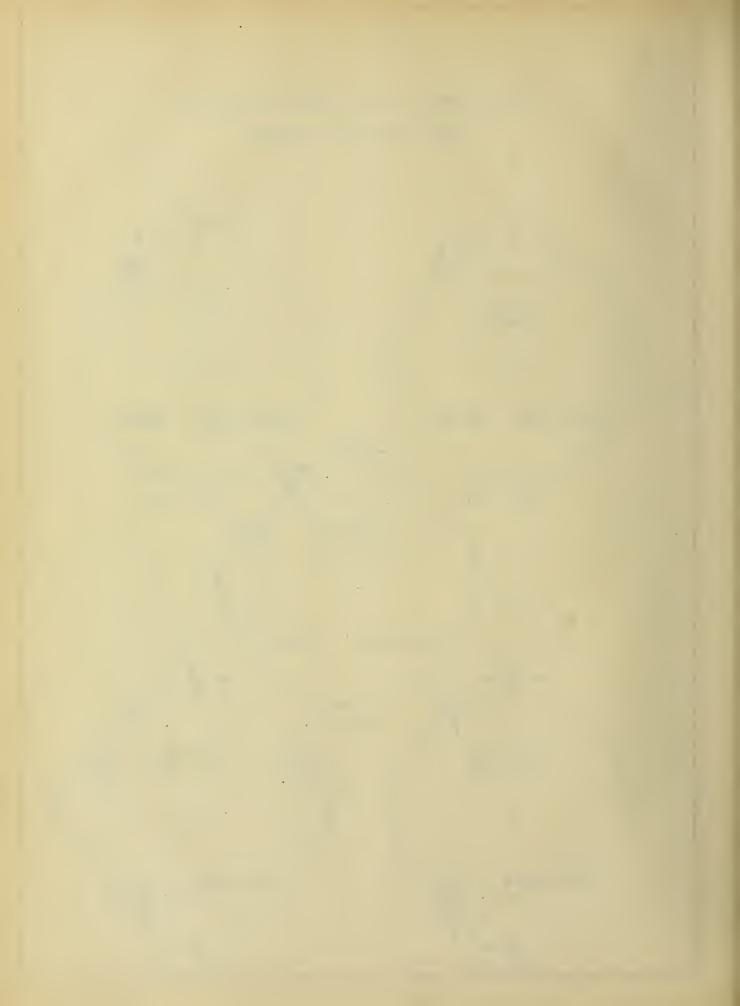




Dimensions and Efficiency of Keyways.

($2\frac{1}{2}$ cold rolled shafts)





(15)
Actual Data.
Test No. 1.
Scale Readings.

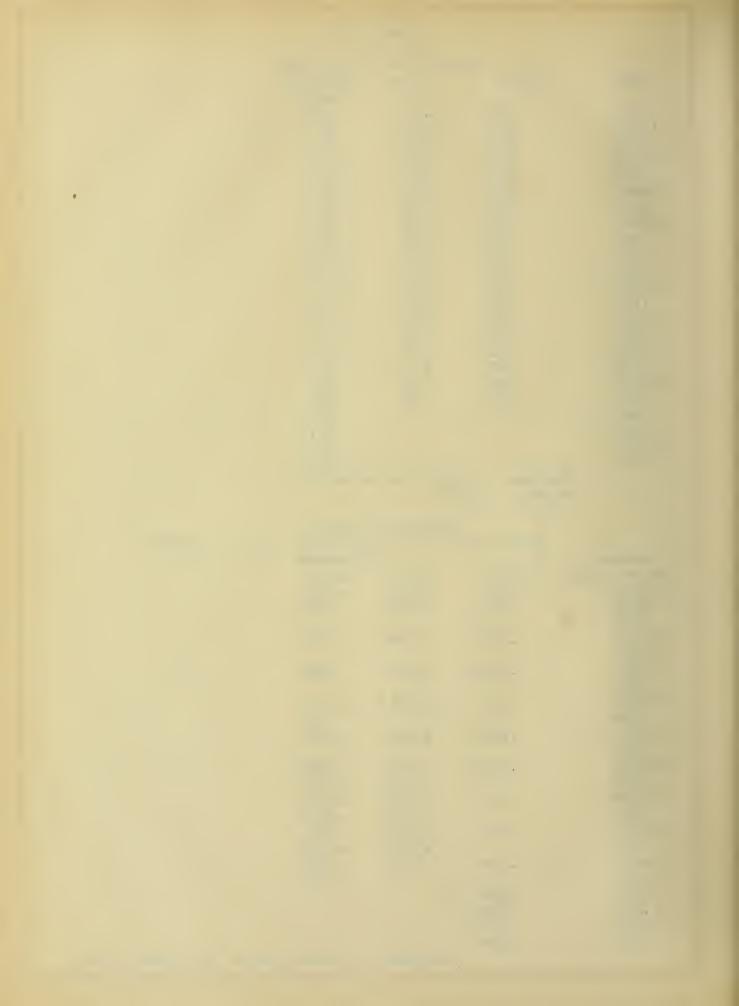
		Actual Scale	e Reading
Load	Solid	Deep	Standard
			Long
1700	138	173	69
10500	130	163	83
1700	138	173	69
20000	121	152	110
1700	138	173	69
29200	112	141	129
1700	138	173	69
36300	107	131	1.45
1700	138	173	69
43000	101	120	161
1700	138	173	69
46300	98	112	171
1700	138	168	72
49000	97	. 105	180
1700	138	163	73
51500	92.5	91	193
1700	138	153	86
53800	91	75	210
1700	138	148	88
61900			80
1700			131.5
66000			66
1700			130.5
	Management	goood inch	

Maximum -- 80000 inch-pounds Twist -- 2600

Computed Results.
Deflections in degrees per inch of length

Maximum -- 50900 pounds per square inch.

	Deflections	in deg	rees per	inch	of length
Stress			Standard		
Outer Fiber	Solid	Deep	Long		
1080	. 000	.000	. 000		
6700	.031	.0387	. 0503		
1080					
12700	. 0658	.0735	.159		
1080					
18600	.1006	.124	.232		
1080					
23100	.12	.163	.294		
1080					
27400	.143	.205	. 356		
1080					
29500	.155	.236	.395		
1080		.0194	.0116		
31200	.159	. 263	. 43		
1080		.0387	.0155		
32800	.178	.317	.441		
1080		.0774	.0581		
34300	.182	.379	.546		
1080			.0736		
39400	. 224				
1080	.0252				
42000	.378				
1080	.029				



(16) Actual Data. Test No. 2.

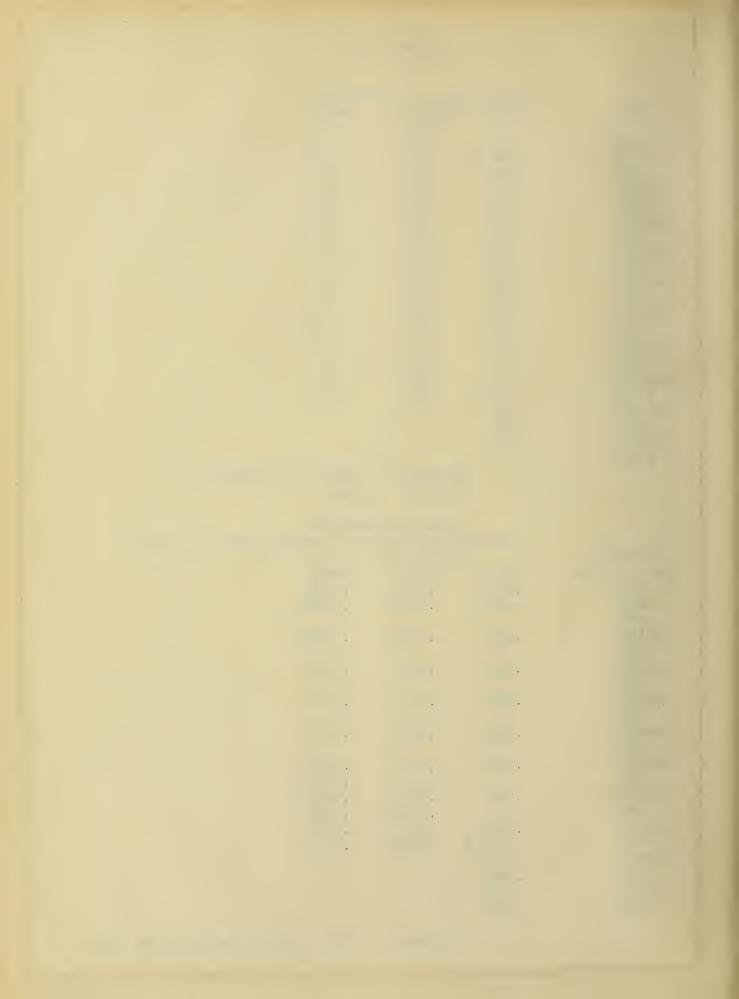
	Actual	Scale Res	dings.
Load	Solid	Standard	Deep
		Long	•
1000	185	163	0
8600	176.5	153	10
1000	185	163	0
18000	170	143	20
1000	185	163	0
26500	162	133	30
1000	185	163	0
35800	155	123	40
1000	185	163	0
43800	145	113	50
1000	185	163	0
53000	135	103	61
1000	185	163	1
58600	135	93	76
1000	185	158	10
60200	133	183	128
1000	184.5	143	57
64100	123		
1000	179		
66700	114		
1000	170		00000
		Marimin	_ 20000

Maximum -- 80000 inch*pounds.
Twist -- 340

Computed Results.

	Deflecti	ons in (degrees p	er inch	of length
Stress	Sta	andard			
Outer Fiber	Solid :	Long	Deep		
637	.000	.000	.000		
5500	.0329	.0387	.0387		
637					
11400	.058	.0774	.0774		
637					
16900	• 089	.116	.116		
637	****				
22800	.166	.155	.155		
637	****	1200	*****		
27900	.155	.194	.194		
637	4 200	4203	4201		
33800	.194	.232	. 236		
637	* 104	• 202	.00387		
37300	.194	271	.294		
	• 134	.271			
637	0.03	.0194	. 0387		
38300	. 201	.461	.495		
637	.00193	.0774	.221		
40800	. 24				
637	.0232				
42500	.275				
637	• 0 58				

Maximum -- 50900 pounds per square inch.



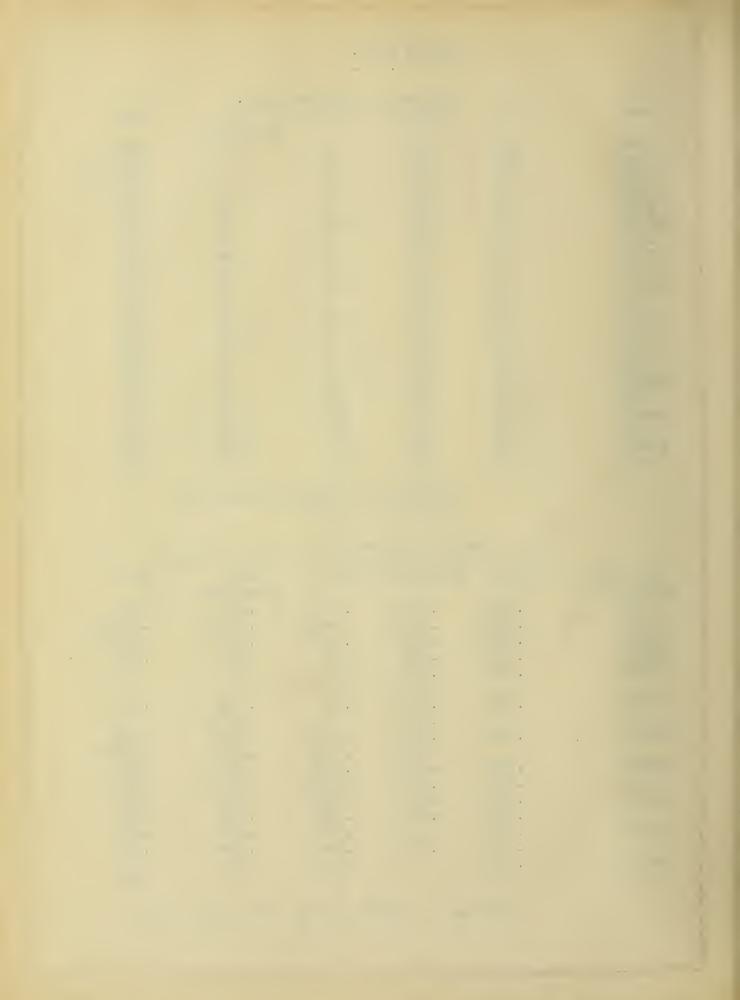
(17) Actual Data. Test No. 3.

		Actual	Scale Re	adings.	
Load	Solid	Standard	Round	One	Flat
				Direction	
1000	178	143	1	- 10	194
12500	168	130	12	0	183
1000	178	143	1	- 10	194
24000	158	119	25	11	172
1000	178	143	1	- 10	194
33 000	149	108	32	22	162
1000	178	143	1	- 10	194
40000	144	100	40	29	155
1000	178	141	1	- 10	194
48000	137	90	50	37	147
1000	178	139	1	~ 8	194
55000	130	75	60	48	137
1000	178	132	6	5	192
60000	123	53	77	61	125
1000	175	115	18	3	184
63000	116	14	103	79	108
1000	171	83	41	17	171
65500	108	off	144	104	83
1000	160	20	76	40	145
68000	97	off	off	137	47
1000	157	Off	135	70	119

Maximum - 91500 inch-pounds Twist - 9200

Computed Results. Deflections in degrees per inch of length. Stress Solid Standard Round One Flat Outer Fiber Direction .000 .000 637 .000 .000 .000 .0387 .0387 7950 . 0503 .0425 .0426 .093 15400 .0774 .093 . 0813 .0852 21000 .112 .135 .12 .124 .124 .166 25500 .132 .151 .151 .151 .00774 30600 .182 .159 .182 .205 .19 .0155 .00774 .228 .224 35000 .186 . 263 .221 .0425 .0194 .00774 .0194 38200 .213 .348 .294 . 275 .267 .108 .0116 . 066 . 0503 .0387 40200 .50 .344 .24 .395 . 294 .0271 .271 .151 .105 .089 41700 . 441 . 430 .272 . 553 .07 .476 .19 .29 .194 43300 .57 .314 .519 .57 .29

Maximum -- 58250 pounds per square inch.



(18) Actual Data. Test No. 4.

		Actual S	cale Read	dings.	
Load	Solid	Standard	Round	One	Flat
				Direction	
1000	140	195	8	25	170
12500	130	184	20	36	161
1000	140	196	8	26	171
22000	121	173	30	46	152
1000	140	196	8	26	171
30500	113	164	39	54	143
1000	140	195	8	26	171
39500	105	155	48	63	134
1000	140	195	9	26	171
46500	100	140	57	71	127
1000	140	194	11	26	170
54500	91	123	71	82	115
1000	138	185	17	31	167
58000	86	112	83	90	106
1000	137	173	20	35	162
62500	80	75	106	108	90
1000	134	141	44	48	151
66000	71	70	145	130	67
1000	129	77	77	67	131
66500	62				26
1000	131		147	102	94

Maximum - 84400 inch-pounds Twist - 410

Computed Results.

Compation Results						
	Deflection	s in degre	es per	inch of length		
Stress	Solid	Standard	Round	One	Flat	
Outer Fiber				Direction		
637	.000	.000	.000	.000	.000	
7950	. 0387	.0426	. 0465	.0426	.0348	
14000	. 0735	.0852	.0852	.0735	. 062	
19400	.105	.12	.12	.112	.104	
25800	.136	.155	.155	.128	.143	
			.00387	•		
29600	.155	.213	.19	.178	.167	
		.00387	.0116			
34700	.19	.279	.244	.221	.213	
	.00774	.0387	.0348	.0232	.0116	
36900	.209	.321	.29	.252	.248	
	.0116	.0852	. 0465	.0387	.031	
39800	.232	.465	.38	.321	.31	
	.0232	.209	.139	.089	.0735	
42000	.267	.484	. 53	.406	.399	
	.0426	.457	.457	.143	.112	
42300	.302				.558	
	.0271			. 298	.394	

Maximum -- 53800 pounds per square inch.



	(1;	"	
Actu	al	Da	ta.
Test	No		5.

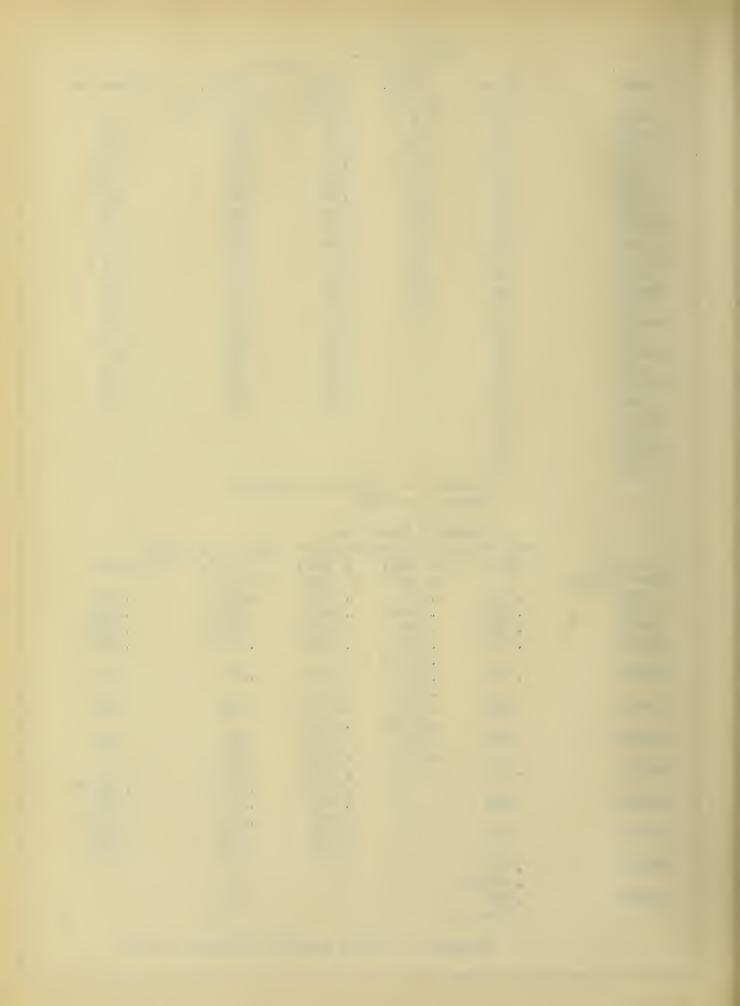
-		Actual	Scale Re	adings.	
Load	Solid	4 Std.	2 Std.	2 Std.	Standard
	7*long	at 90°	at 180°	at $90^{\circ}(8\frac{1}{2})$	
1000	10	9	195	169	5
9000	16	23	183	158	13
1000	10	9	195	169	5
15000	21	37	176	151	19
1000	10	9	195	169	5
21000	25	47	168	144	27
1000	10	11	195	169	5
28000	31	64	160	135	35
1000	10	15	195	169	5
34000	35	90	151	127	41
1000	10	29	193	167	5
38500	38	134	143	120	46
1000	10	65	190	167	5
44000	42		130	110	54
1000	10		185	163	6
47500	46		108	94	61
1000	10		169	153	8
52000	49		77	73	70
1000	10		141	137	12
56500	54				
1000	11				
61000	61				
1000	15				
	Mari	1 0 1 m	and inch	mannda	

Maximum -- 81000 inch-pounds Twist -- 8900

Computed Results.

	Deflecti	on in deg	rees per	inch of leng	gth
Stress	Solid			2 Std.	Standard
Outer Fiber			at 180°	at 90°	
637	• 000	.000	.000	.000	.000
5700	.0232	.0542	.0465		. 031
9500	. 0426	.108			. 0542
13400	• 058	.147	.1045	. 097	. 0852
		.00774			
17600	.0814	.213	.135	.132	.116
03.000	00.0	. 0232	2 70 0	2.05	170
31600	.093	.314	.170	.163	. 139
04500	3.00	.0774	.00774	3.0	3.50
24500	.108	.523	.201	.19	.159
28000	.124	.217	.0194	.0774	.19
20000	* TO.		.0774	. 0232	.00387
30200	.139		.337	. 29	.216
00200	* 100		.101	.062	.0116
33100	.151		457	.372	.252
	4-0-		. 209	.124	. 0271
36000	.17				
	.00774				
38800	.198				
	.0194				

Maximum -- 51600 pounds per square inch.



		(00)				
		(20)				
Actual Data.						
		Test No.	6.			
			ial Scale	Readings		
	0.111				Ohan da ad	
Load	Solid	4 Std.	2 Std.	2 Std.	Standard	
		at 90°	at 180°	at 90°		
1000	145	172	_ 4	10	191	
13000	135	149	11	27	179	
1000	145	172	· 4	10	191	
16000	132	143	15	25	175	
1000	145	172	·- 4	10	191	
22000	126	130	23	34	169	
1000	145	172	<u> </u>	10	191	
28500	121	109	32	43	162	
			- 4	10	191	
1000	145	163				
36500	113	31	45	5 5	153	
1000	145	108	- 1	10	190	
40500	110	Off	53	63	144	
1000	145	5	3	11	190	
44000	107		65	70	145	
1000	145		10	15	189	
48000	104		87	85	140	
1000	145		27	27	187	
52000	100		166	132	123	
1000	145		98	66	183	
			•		118	
56000	95					
1000	144			133	176	
59000	90				94	
1000	141				157	
2000			reaco in ah	mannda	****	
		ximum '	76000 inch	-pounds		
	T w	ist '	79°0°			
	C	omputed Re	anlta			
					n 4.7s	
			egrees per			
Stress	Solid	4 Std.	2 Std.	2 Std.	Standard	
Outer Fi		at 90°	at 1800	at 90°		
637	.000	.000	.000	.000	000	
					.000	
8300	. 0387	. 089	. 058	.0658	. 0465	
10400	. 0502	.108	. 0735	. 058	.0619	
14000	.0735	.167	.104	.093	. 085	
18300						
19900	, 093	. 244	.139	.128	.112	
		.0348				
23200	.128	. 545	.190	.174	.147	
	4 2 14 4	. 248	.0116		-	
00000	3.65			005	3.00	
25800	•135	. 625	.221	. 205	.182	
			.0271	.0038	.00387	
28000	.147		.267	.232	.178	
20000	0 4 7 1					
			. 054	.0193	.00774	
30600	.159		.352	.29	.197	
			.12	.0658	. 0155	
33100	.174		4 3.00	.29	.263	
207.00	* 1 1 4					
				.0217	.031	
35700	.193				. 282	

Maximum -- 48400 pounds per square inch.

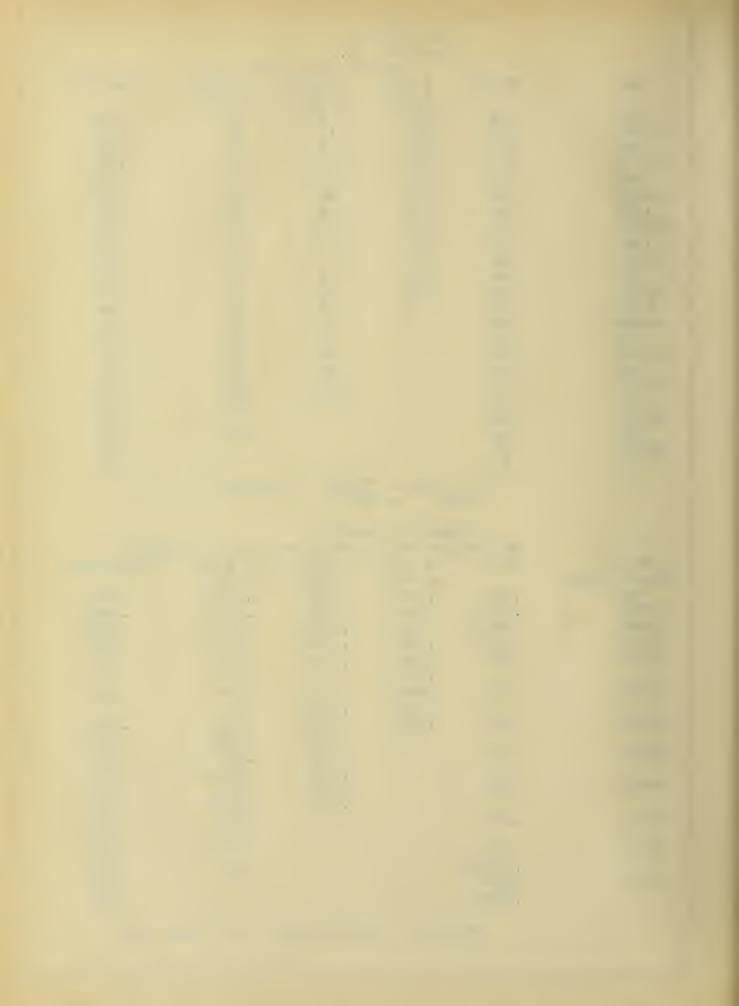
.48

.193 .00387 .213 .0155

35700

37600

. 282 . 058 . 375 . 135



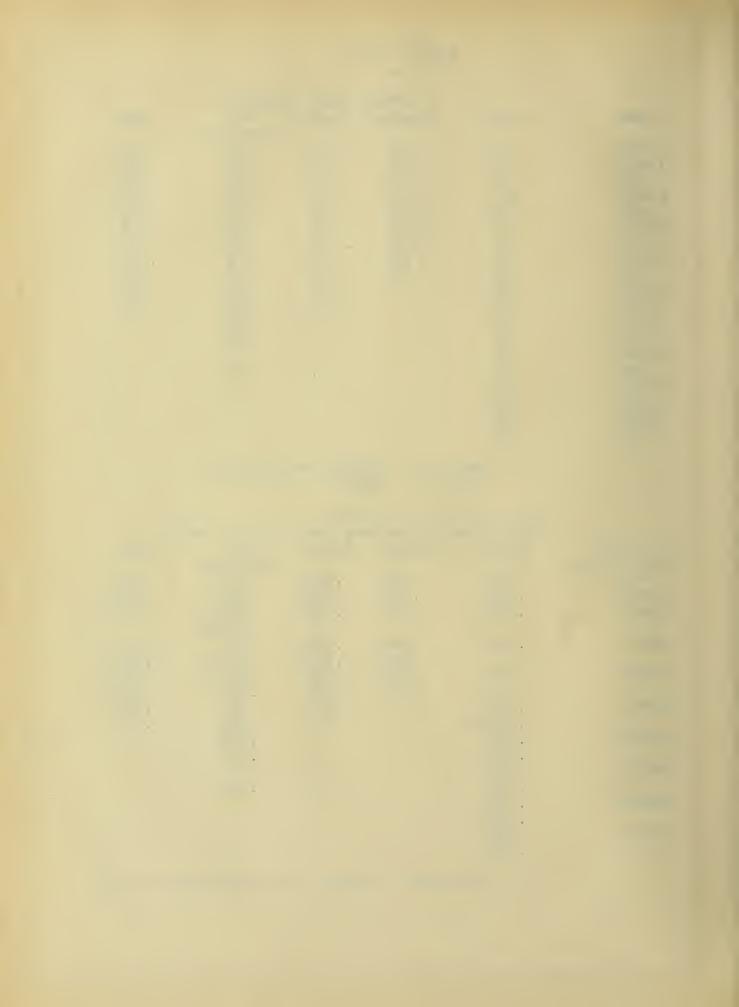
(21)
Actual Data.
Test No. 7.

Load	Solid	Actual Standard		Readings.	Flat
1000 12000 1000 22000 1000 30000 1000 32500 1000 33800 1000 35900 1000 35500 1000 36500 1000	2 12 22 21 28 28 2 30 3 3 3 3 3 3 3 9 46 15 80 38	167 155 167 143 167 130 161 - 15 15	- 7 4 - 7 15 - 7 26 - 4 60 30 143 110	Direction 189 178 188 167 188 158 144 173 127 159 36 68	144 132 144 122 144 113 141 97 127 75 108

Maximum -- 81800 inch-pounds. Twist -- 2250°

Computed Results. Deflection in degrees per inch of length Stress Solid Standard Round One Flat Outer Fiber Direction .000 .000 .000 700 .000 .000 .0387 . 0426 . 0425 . 093 . 085 .0464 8400 .0435 15400 .0737 . 085 .085 .00387 .143 .128 .12 .12 21000 .101 . 0232 .0116 .0116 .0155 22800 .109 .705 . 259 .174 .182 . 552 .143 .0619 .0658 23700 .120 .24 .267 .576 .00387 .116 .139 23800 .592 .136 .468 .0228 24500 .145 .0271 .646 24900 .171 .0502 .302 25600 .139

Maximum -- 57300 pounds per square inch.



(22) Actual Data. Test No. 8.

Load	Solid	Actual Standard	Scale Round	Reading. One	Flat
1000 11500 1000 17000 1000 21700 1000 25000 1000 38000 1000 32000 1000 33500 1000 34000 1000 36500	160 150 160 144 158 140 158 137 157 134 157 130 156 128 156 123 152 120 149	9 21 9 26 9 33 10 36 10 40 11 51 30 87 54	Round 194 183 194 177 193 172 193 167 192 164 192 153 183 139 170	One Direction 4 13 4 18 4 24 24 4 28 5 32 5 40 12 47 17 89 56 133 99	139 128 139 123 138 118 137 111 137 104 133 98 128 45 79
1000	91 123				

Maximum -- 83300 inch-pounds Twist -- 2470°

Computed Results.

	0 0 11.1				
	Deflection			nch of length	
Stress	Solid	Standard	Round	One	Flat
Outer Fiber				Direction	
700	.000	.000	.000	.000	.000
8050	. 0387	. 0464	. 0425	.0348	. 0425
11900	.0619	.0658	.0658	. 0542	. 062
15200	.0774	.0928	.085	.0774	.0813
		.0038	.0038		.0038
17500	. 089	.100	.105	. 0928	.100
	.0116	.0038	.0077		.0077
19600	.1008	.12	.116	.108	.108
	.0077	.0077	.0038	.0077	
21700	.116	.138	.159	.139	.135
	.0155	.0812	.0425	.031	.0232
22400	.124	.32	.213	.167	.159
	.0155	.174	.0928	.0504	.0425
23500	.143		,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	.329	.364
	.031			.215	.232
23800	.155			.500	
	.0812			.368	
25600	.267			1000	
20000	4 900 8				

Maximum -- 58300 pounds per square inch.



(23)
Actual Data.
Test No. 9.

	Actual	Scale Read	ings.
Load	Solid	Standard .	Deep
		Long	
1000	000	11	7
15000	4.5	17	13
38000	12	25	21
56700	18	32	28
65000	21	35	34
75000	24	39	37
87000 .	27	44	43
93500	29	48	47
101500	32	54	51
107500	34	59	66
119500	41	87	115
122000	44	109	160
125500	46	118	175
135000	61		

Maximum -- 183000 inch-pounds Twist -- 61500

Computed Results.

Deflection in degrees per inch of length Solid Standard Deep Stress Outer Fiber Long .000 326 .000 .000 5900 .0174 . 0232 .0232 .0464 . 054 . 054 12400 18200 . 0695 .081 .081 21200 .0925 .081 . 1.04 .097 .108 .116 24400 .108 28400 .104 .139 30500 .112 .143 .154 .124 .166 .170 32900 .131 . 185 35000 . 228 .158 39000 . 294 . 417 39800 .17 .378 .591 .178 41000 .413 .65 44000 . 236

Maximum -- 59700 pounds per square inch.



(24) Actual Data. Test No. 10.

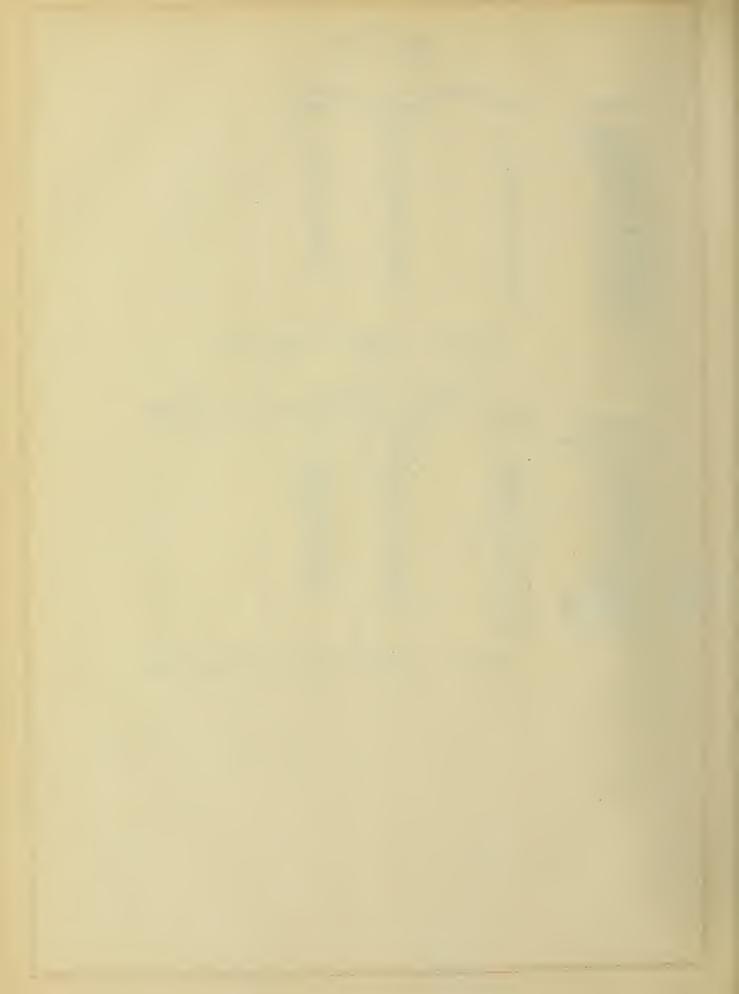
	Act	ual Scale Re	eadings.
Load	Solid	Standard	Deep
		Long	
1000	5	000	7.5
25000	13	8	17
42500	17	15	24
58000	23	21	31
81600	30	29	40
96500	35	37	49
104000	38	43	58
113500	42	5 5	80
125000	49	86	145
132500	55	148	
138500	68		
146000	103		

Maximum -- 188000 inch-pounds Twist -- 790

Computed Results
Deflection in degrees per inch of length

Deflec.	tion in degr	ees per	inch of	length
Solid	Standard	Deep		
	Long			
.000	.000	.000		
.0293	.0310	.0386		
.0463	.058	.0656		
.0695	.0811	. 0887		
.0965	.102	.127		
.116	.143	.162		
.127	.166	.197		
.143	.212	.274		
.170	.332	.533		
.193	.572			
.243				
.376				
	Solid .000 .0293 .0463 .0695 .0965 .116 .127 .143 .170 .193 .243	Solid Standard Long .000 .0293 .0310 .0463 .058 .0695 .0811 .0965 .102 .116 .143 .127 .166 .143 .212 .170 .332 .193 .572 .243	Solid Standard Deep Long .000 .000 .0293 .0310 .0386 .0463 .058 .0656 .0695 .0811 .0887 .0965 .102 .127 .116 .143 .162 .127 .166 .197 .143 .212 .274 .170 .332 .533 .193 .572 .243	Long .000 .000 .000 .0293 .0310 .0386 .0463 .058 .0656 .0695 .0811 .0887 .0965 .102 .127 .116 .143 .162 .127 .166 .197 .143 .212 .274 .170 .332 .533 .193 .572

Maximum -- 61300 pounds per square inch.



(25) Actual Data. Test No. 11.

Load Solid STandard 2 Std. 2 Std. 4 Std. at 90° at 180° at 18		~ `				
1000 - 8 12 179 173 33 17500 - 3 18 173 167 42 1000 - 8 12 178 173 33 29000 0 22 168 161 47 1000 - 8 12 178 173 33 40000 4 26 163 158 54 1000 - 8 12 178 173 33 50000 7 30 159 154 60 1000 - 8 12 178 173 34 62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 <			Actual :			
1000 - 8	Load	Solid	STandard	2 Std.	2 Std.	4 Std.
1000 -8 12 179 173 33 17500 -3 18 173 167 42 1000 -8 12 178 173 33 29000 0 22 168 161 47 1000 -8 12 178 173 33 40000 4 26 163 158 54 1000 -8 12 178 173 33 50000 7 30 159 154 60 1000 -8 12 178 173 34 62500 11 35 155 148 69 1000 -8 12 178 173 36 72500 14 39 150 144 83 1000 -8 12 178 173 43 83000 17 42 145 139 108 1000 -8 12 178 172 62 91000 20 47 <td></td> <td></td> <td></td> <td>at 900</td> <td></td> <td>at 900</td>				at 900		at 900
17500 - 3 18 173 167 42 1000 - 8 12 178 173 33 29000 0 22 168 161 47 1000 - 8 12 178 173 33 40000 4 26 163 158 54 1000 - 8 12 178 173 33 50000 7 30 159 154 60 1000 - 8 12 178 173 34 62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8	1000	Ω	12	179		22
1000 - 8 12 178 173 33 29000 0 22 168 161 47 1000 - 8 12 178 173 33 40000 4 26 163 158 54 1000 - 8 12 178 173 33 50000 7 30 159 154 60 1000 - 8 12 178 173 34 62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14						
29000 0 22 168 161 47 1000 - 8 12 178 173 33 40000 4 26 163 158 54 1000 - 8 12 178 173 33 50000 7 30 159 154 60 1000 - 8 12 178 173 34 62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 </td <td></td> <td></td> <td></td> <td></td> <td></td> <td></td>						
1000 - 8 12 178 173 33 40000 4 26 163 158 54 1000 - 8 12 178 173 33 50000 7 30 159 154 60 1000 - 8 12 178 173 34 62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 105000 24 53 12		- 8				
40000 4 26 163 158 54 1000 -8 12 178 173 33 50000 7 30 159 154 60 1000 -8 12 178 173 34 62500 11 35 155 148 69 1000 -8 12 178 173 36 72500 14 39 150 144 82 1000 -8 12 178 173 43 83000 17 42 145 139 108 1000 -8 12 178 172 62 91000 20 47 139 132 176 1000 -8 14 175 170 123 99500 22 50 134 127 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 <	29000		22	168		
40000 4 26 163 158 54 1000 -8 12 178 173 33 50000 7 30 159 154 60 1000 -8 12 178 173 34 62500 11 35 155 148 69 1000 -8 12 178 173 36 72500 14 39 150 144 82 1000 -8 12 178 173 43 83000 17 42 145 139 108 1000 -8 12 178 172 62 91000 20 47 139 132 176 1000 -8 14 175 170 123 99500 22 50 134 127 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 <	1000	- 8	12	178	173	3 3
1000 - 8 12 178 173 33 50000 7 30 159 154 60 1000 - 8 12 178 173 34 62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41	40000	4	26	163	158	54
50000 7 30 159 154 60 1000 -8 12 178 173 34 62500 11 35 155 148 69 1000 -8 12 178 173 36 72500 14 39 150 144 82 1000 -8 12 178 173 43 83000 17 42 145 139 108 1000 -8 12 178 172 62 91000 20 47 139 132 176 1000 -8 14 175 170 123 99500 22 50 134 127 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41		- 8				
1000 - 8 12 178 173 34 62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
62500 11 35 155 148 69 1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41		-				
1000 - 8 12 178 173 36 72500 14 39 150 144 82 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
72500 14 39 150 144 83 1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
1000 - 8 12 178 173 43 83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
83000 17 42 145 139 108 1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41	1000	8	12	178	173	43
1000 - 8 12 178 172 62 91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41	83000	17	42	145	139	108
91000 20 47 139 132 176 1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41	1000	- 8		178	172	62
1000 - 8 14 175 170 123 99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
99500 22 50 134 127 off 1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
1000 - 8 15 172 166 off 105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
105000 24 53 127 121 115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						
115000 26 57 114 107 117500 29 65 88 83 123000 32 73 44 41						011
117500 29 65 88 83 123000 32 73 44 41						
123000 32 73 44 41						
		29				
	123000	32	73	44	41	
127300 35						

Maximum -- 169000 inch-pounds Twist -- 6850

Maximum -- 55100 pounds per square inch.

Computed Results. Deflection in degrees per inch of length. Solid 2 Std. 4Std. Stress Standard 2 Std. at 90° at 180° at 90° Outer Fiber 326 .000 .000 .000 .000 .000 .0348 5700 .0193 .0232 .0232 .0232 9400 .0309 .0386 .054 . 0425 .0463 13000 .0463 .054 .0617 .0579 .081 16300 . 0579 .0695 .0772 .0733 .104 .00386 20400 .0734 .0887 .0926 .139 .0965 .0116 .189 23600 .085 .104 .112 .112 .0386 27100 .0965 .116 .131 .131 .29 .112 29700 .552 .108 .135 .155 .158 .0772 .0155 .0116 .348 32400 .116 .147 .174 .177 .0116 .0232 .027 34200 .124 .239 .158 .239 .131 36300 .174 .251 .253 38300 .143 .205 .351 .348 40100 .155 .236 .521 .510 .0772 .0656 .324 .298 41500



(26) Actual Data. Test No. 12.

4 Std.
at 900
1
11
24
34
40
60
103
off

Maximum -- 165000 inch-pounds Twist -- 730°

Computed Results.

Oomparoa Robartos						
	Deflecti	on in degre	es per in	ich of len	gth	
Stress	Solid	Standard	2 Std.	2 Std.	4 Std.	
Outer Fiber			at 90°	at 180°	at 90°	
3260	.000	.000	.000	.000	.000	
6400	.0155	.0232	. 027	.0232	. 0386	
14300	. 0463	. 0579	. 0656	.0579	.0887	
18900	.0656	.0772	.0887	.0772	.127	
21100	. 0733	. 0887	.0965	.0887	.151	
24800	.0887	.104	.120	.112	. 228	
27700	.100	.12	.143	.131	.394	
31200	.112	.135	.166	.155		
33400	.120	.147	.197	.189		
35800	.131	.166	. 255	.247		
39100	.147	.205	. 436	. 44		
40900	.162	.243	.68			
42500	.178	.301				
44300	. 205	. 409				
45600	.244					

Maximum -- 53800 pounds per square inch.



(26)
Actual Data.
Test No. 13.

		Actual	Scale	Readings.	
Load	Solid	Standard	Round	One	Flat
				Direction	
1000	19	5.5	188	178	15
20000	25	12	183	172	21.5
44500	32	21	175	164	30
59500	37	27	170	158	35
80500	44	35	162	151	43
96500	50	42	156	145	48
110500	55	51	149	138	56
121000	61	65	140	139	64
131500	71	104	120	110	83
137500	82	184	84	80	117
143300	91		42	45	156
147700	115				
150600	140				

Maximum — 198000 inch-pounds Twist — 14400

Computed Results. Deflection in degrees per inch of length Standard Round Flat Stress Solid One Direction Outer Fiber 326 .000 .000 . 000 .000 .000 .0232 6500 .0232 .0193 .0232 .0251 14500 .0502 .068 .0502 . 054 .0579 .0695 19900 . 0695 .0772 .0772 . 081 .1000 .112 26600 . 0965 .104 .108 31400 .114 .139 .123 .127 .127 .139 .154 .158 36100 .174 .151 .162 .189 .228 .185 39500 .189 . 262 42800 .201 .378 . 263 . 262 44800 .243 .687 .402 . 355 .394 .278 46700 . 564 .513 .545 48300 .370 48200 . 467

Maximum -- 64500 pounds per square inch.



(27) Actual Data. Test No. 14.

		Actual	Scale R	leadings.	
Load	Solid	Standard	Round	One	Flat
				Direction	
1000	20	10	177	182	0
23700	27	18	169	175	8
44700	34	26	162	169	15
63500	3 9	32	156	163	22
77000	45	37	152	158	26.5
87000	48	37	148	155	30
94000	50	45	145	152.5	32.5
106500	55	52	140	147	38
118500	60	64	132	140	45
126000	66	82	121	132	54
131700	72	111	108	120	66
139600	81	179	80	99	89
143700	94	off	37	67	124
147600	111	off	off	13	184
150800	136				

Computed Results.

	Comp	Jurea VeenT	U 🖰 🐞		
	Deflection	in degrees	per inch	of length	
Stress	Solid	Standard	Round	One	Flat
Outer Fibe	r		Di	rection	
326	.000	.000	.000	.000	۰ 000
7750	.027	.0309	.0309	. 027	.0309
14600	. 054	.0618	.058	. 0502	. 058
20700	.0734	. 085	.081	.0734	。085
25300	. 0965	.104	. 096	.0927	.102
28400	.108	.12	.112	.104	.116
30600	.116	.135	.123	.116	.125
34700	.135	.162	.143	.135	.147
38700	.154	.208	.174	.162	.174
41200	.177	.278	.216	.193	.208
43000	.201	.39	. 266	.239	. 255
45800	.236	.652	.374	. 32	. 344
46800	.286		.540	. 434	.479
48300	.351			.653	.71
	.448				



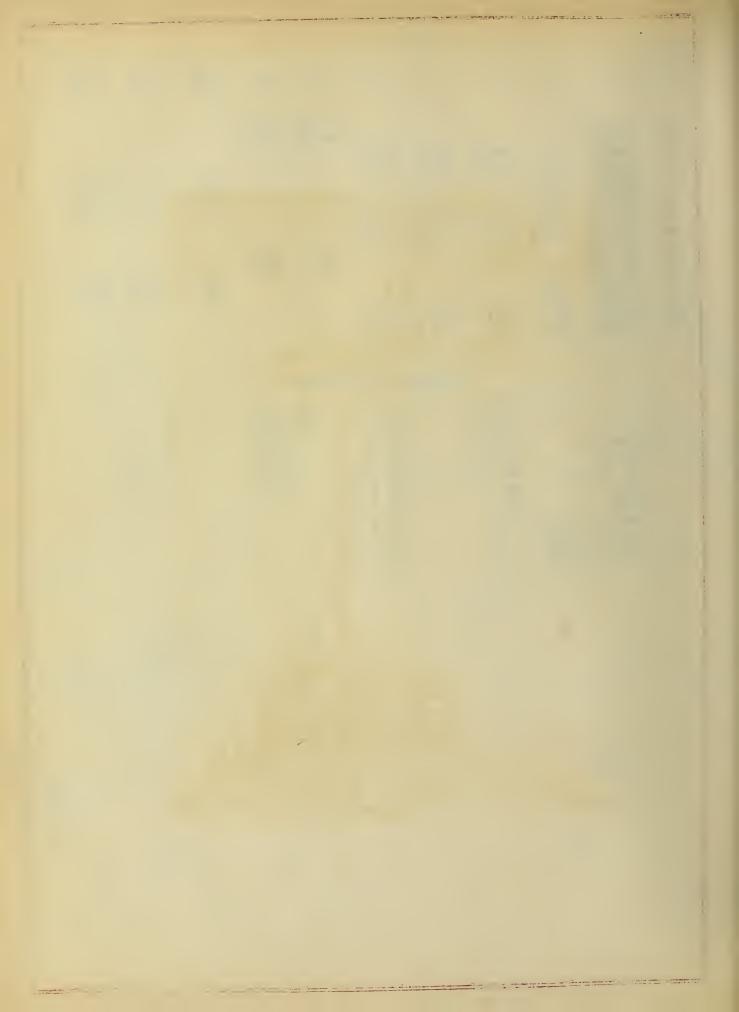
No. of Test	Solid	Deep	Standar Long	đ	
1 2 9 10	36700 36700 36800 37000	27800 35000 31400 29600	30200 35000 30500 30000		
No. of Test	Solid	Standard	Round	One Directi	on Flat
3 4 7 8 13 14	36600 36900 21800 19500 36300 37500	31000 29000 19800 18100 31400 31700	31900 29700 19100 18200 33500 34300	32500 32600 19500 18200 33400 34000	32000 19400 17900 33800
No. of Test	Solid	Standard	2 at	2 at 180°	4 at 90°
5 6 11 12	35400 37100 37800 37900	27800 29800 33400 33500	26900 27400 29000 28100	26200 22000 27900 26800	17000 18200 19400 19100

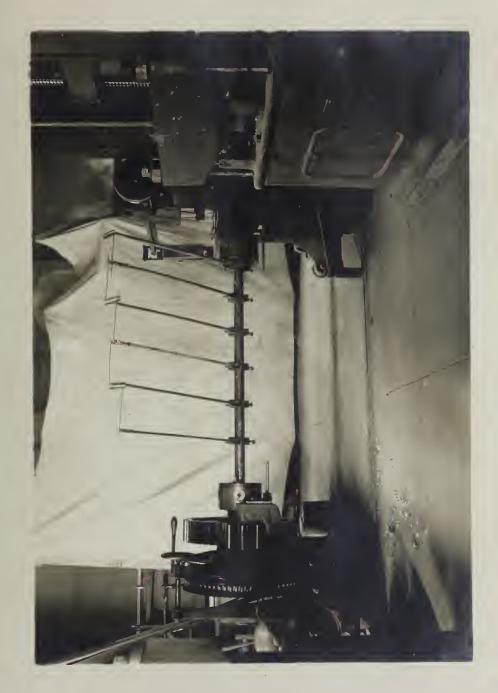


Wa af	(29)									
No. of Test	Solid	Std.	Rd.	O.D.	Flat	Deep	Long	2 8t	2 at 180°	4 at 900
1 2 3 4	36700 36700 36600 36900	84.5 78.5	86.9 80.4	88.5 88.3	90.5 86.6	78.5 95.4	81.3 95.4			
4 5 6 7 8 9	35400 37100 21800 19500	78.5 80.3 90.8 92.8	87.5 93.3	89.3	88.9 91.7			76.0 73.8	74.0 59.3	48.0
9 10 11	36800 37000 37800	88.4	00.0		V-4-1	85.3 80.0	82.8 81.0	76.7	73.9	51.3
12 13 14	37900 36300 37500	88.4 86.5 84.5	92.3 91.4	92.0 90.6	93.1 92.7			74.2	70.7	50.4

Average Efficiency.

	Cold	Rolled 21 "	Turned 1-15/16"
Standard	80.4	86.9	91.8
Round	83.6	91.8	90.4
One Direct.	88.4	91.3	91.3
Flat	88.5	92.9	90.3
Deep	86.9	82.6	
Long	88.3	81.9	
2 Std. at 90°	74.9	75.4	
2Std.at180°	66.6	72.3	
4 Std.at90°	48.5	50.8	





Arrangement of Apparatus





Brittle Ductile
Typical Fractures



